

REAL CODED GA FOR TUNING OF SEMI-ACTIVE RAILWAY VEHICLE
SUSPENSION SYSTEM

ARASH BAGHERI

A project report submitted in fulfilment of the
requirements for the award of the degree of
Master of Engineering (Mechanical)

Faculty of Mechanical Engineering
Universiti Teknologi Malaysia

JUNE 2015

To my beloved mother and father

ACKNOWLEDGEMENT

I wish to express my sincere appreciation to my supervisor Prof. Dr. Hishamuddin Jamaluddin for his encouragement, guidance and critics. Without his continued support and interest, this thesis would not have been the same as presented here.

Lastly, I also wish to extend my thanks to everyone who have helped directly or indirectly in executing this master's project. My deep gratitude goes to my beloved family for their constant love, unlimited support and understanding.

ABSTRACT

To maintain a high level of comfort expected by passengers from transportation vehicle while maintaining a high safety standards railway vehicle suspension system contribute the most significant impact. The main requirement of a vehicle suspension is that, it should be able to minimize the vertical displacement and the acceleration of the body in order to improve passenger comfort. A viable alternative to maintain the level of comfort is to use a semi-active suspension system with magneto-rheological (MR) damper which will reduce the inherent tradeoff between the ride comfort and road holding characteristic of the vehicle. Since the behavior of semi-active devices is often highly nonlinear, one of the main challenges in the application of this technology is the development of appropriate control system. In this thesis, the development of a semi-active suspension control of half car model of railway vehicle using stability augmentation control system is studied. A mathematical modelling and computer simulation model of secondary half car semi-active suspension controller algorithm have been developed within Matlab-SIMULINK. The tuning of this controller was developed by using Genetic Algorithm (GA).

ABSTRAK

Untuk mengekalkan tahap keselesaan yang tinggi yang diharapkan oleh penumpang dari kenderaan pengangkutan di samping mengekalkan tahap keselamatan yang tinggi, sistem gantungan menyumbang dengan paling ketara. Keperluan utama sistem penggantungan kenderaan adalah, ia mestilah mampu untuk mengurangkan anjakan dan pecutan badan menegak / melintang untuk meningkatkan keselesaan penumpang. Satu alternatif yang berdaya maju untuk mengekalkan tahap keselesaan adalah dengan menggunakan sistem sgantungan separa-aktif dengan peredam magneto-reologi (MR) yang akan mengurangkan keseimbangan yang wujud antara keselesaan perjalanan dan ciri-ciri yang memegang jalan kenderaan. Oleh kerana kelakuan peredam separa-aktif kebiasaannya sangat tidak linear, salah satu cabaran utama dalam penggunaan teknologi ini ialah pembangunan sistem kawalan yang sesuai. Dalam tesis ini, pembangunan kawalan sistem gantungan separa-aktif model kereta separuh daripada kenderaan keretapi menggunakan kestabilan sistem kawalan pembesaran dikaji. Pemodelan dan simulasi komputer model matematik kereta separuh kedua, algoritma pengawal sistem gantungan separa-aktif telah dibangunkan dalam Matlab-SIMULINK. The penalaan pengawal ini telah dibangunkan dengan menggunakan Algoritma Genetik (GA).

TABLE OF CONTENTS

CHAPTER	TITLE	PAGE
	DECLARATION	ii
	DEDICATION	iii
	ACKNOWLEDGEMENT	iv
	ABSTRACT	v
	ABSTRAK	vi
	TABLE OF CONTENTS	vii
	LIST OF TABLES	xi
	LIST OF FIGURES	xii
	LIST OF SYMBOLS	xv
 1	 INTRODUCTION	 1
	1.1 Research Background	2
	1.2 Objectives	3
	1.3 Problem Statements	3
	1.4 Research Question	4
	1.5 Theoretical Frame Work	4
	1.6 Scopes of Research	4
	1.7 Research Methodology and Flowchart	5
	1.8 Thesis Outline	7
 2	 LITERATURE REVIEW	 9
	2.1 Introduction	9
	2.2 Railway Vehicle Suspension System	9
	2.2.1 Primary suspension	10
	2.2.2 Secondary suspension	11

	2.2.3	Titling	13
	2.2.4	Practical implementation	16
2.3		Classification of Vehicle Suspension Systems	17
	2.3.1	Passive suspension system	17
	2.3.2	Semi-active suspension system	18
	2.3.3	Active suspension	21
2.4		Controller Design for Railway Vehicle	23
	2.4.1	PID controller	23
	2.4.2	Skyhook controller	26
	2.4.3	Fuzzy controller	28
	2.4.4	LQ controller (LQC)	29
	2.4.5	H^∞ controller	30
	2.4.6	Linear quadratic regulator	31
	2.4.7	Neural network	32
	2.4.8	Stability augmentation system	33
2.5		Tuning Methods	34
	2.5.1	Genetic algorithm	34
	2.5.2	Particle swarm optimization (PSO) algorithm	36
2.6		Performance Criteria	38
	2.6.1	Ride quality	38
2.7		Conclusion	43
3		RAILWAY VEHICLE DYNAMIC MODEL	44
	3.1	Introduction	44
	3.2	Level of Railway Vehicle Suspension System	44
	3.2.1	Primary and secondary suspension system	45
	3.2.2	Titling suspension system	46
	3.3	Model of Railway Suspension System	47
	3.3.1	Passive model description	47
	3.3.2	Semi-active suspension system	51
	3.4	Semi-active Suspension Actuator	53
	3.4.1	MR damper system	54
	3.4.2	MR damper parametric model	55
	3.5	Conclusion	57

4	SEMI-ACTIVE SUSPENSION CONTROLLER USING GA	58
4.1	Introduction	58
4.2	SAS Controler Structure in SIMULINK	58
4.2.1	Controller parameters	59
4.3	Controller Parameters Tuning	59
4.3.1	Differences between GA and traditional methods	59
4.3.2	Overview of GA	60
4.3.3	Structure of a GA	60
4.3.4	Representation issue	61
4.3.5	Selection mechanism	62
4.3.6	Recombination through crossover and mutation	64
4.3.7	Applications of Gas	65
4.3.8	Real-coded genetic algorithms (RCGA)	66
4.4	Real-coded Genetic Algorithm for Tuning SAS Controller	66
4.4.1	Real coding	67
4.4.2	Initialise the GA parameters and generate an initial, random population of individuals	67
4.4.3	Evaluate the fitness of each chromosome	68
4.4.4	Genetic operations	69
5	RESULT AND DISCUSSION	76
5.1	Cost Function Selection	76
5.2	GA Parameters	78
5.2.1	Range of parameters	78
5.2.2	Number of iteration	79
5.2.3	Number of population	80
5.2.4	Mutation rate	81
5.3	Comparison Between Semi-active Controler with Passive Controller	82
5.3.1	Body displacement under different inputs	82
5.3.2	Body accelaration under different inputs	86
5.3.3	Body roll angle under different inputs	90
5.3.4	Body roll rate under different inputs	92

6	CONCLUSION	96
----------	-------------------	-----------

	REFERENCES	97
--	-------------------	-----------

LIST OF TABLES

TABLE NO.	TITLE	PAGE
2.1	Ride evaluation scales	41
2.2	British railway ride quality and ride index	42
3.1	Passive suspension system parameters identification	48
3.2	Passive suspension system parameters	49
3.3	Model parameters	49
3.4	Semi-active suspension system parameters identification	53
3.5	Semi-active suspension system parameters	53
3.6	Parameters for the MR damper model	56

LIST OF FIGURES

FIGURE NO.	TITLE	PAGE
1.1	Flowchart of methodology	6
2.1	The lateral force is decreased due to tilting of the carbody (right).	14
2.2	Principle of an active anti-roll bar.	15
2.3	Model passive suspension	18
2.4	Model semi-active Suspensions	21
2.5	Model active suspensions	23
3.1	Stages of suspension systems in railway vehicle	45
3.2	Tilting train schematic	46
3.3	Passive suspension system model	48
3.4	SIMULINK modeling for passive suspension systems	50
3.5	Semi-active suspension	51
3.6	SIMULINK modeling for semi-active suspension systems	52
3.7	MR Fluid in suspension model	54
3.8	Bouc-Wen modified MR damper model (Spencer -1997)	55
4.1	SAS controller structure	58
4.2	Structure of a GA	61
4.3	Selection application	63
4.4	Stochastic universal sampling	63
4.5	Initialization step	68
4.6	Fitness evaluation	69
4.7	Probability finding for selectin operator	70
4.8	Population roulette wheel	70
4.9	Single-point crossover operator example	72
4.10	Double-point crossover operator example	72
4.11	Arithmetical crossover operator example	74
4.12	Arithmetical crossover operator example	75

5.1	Peak acceleration by using RMS cost function and absolute cost function	77
5.2	Peak acceleration for 4 different ranges	78
5.3	Peak acceleration of every 10 generation obtained by three times running GA	79
5.4	Peak acceleration obtained by 3 different number of population	81
5.5	Peak acceleration obtained by four mutation rate	82
5.6	Carbody displacement under yellow random input, red passive and green semi-active displacement.	83
5.7	Carbody displacement under yellow step input, red passive and green semi-active displacement.	84
5.8	Carbody displacement under yellow sine wave input at frequency of 1Hz, red passive and green semi-active displacement.	85
5.9	Carbody displacement under yellow sine wave input at frequency of 4Hz, red passive and green semi-active displacement.	85
5.10	Carbody displacement under yellow sine wave input at frequency of 7Hz, red passive and green semi-active displacement.	86
5.11	Carbody acceleration under random input, yellow passive and red semi-active acceleration.	87
5.12	Carbody acceleration under step input, yellow passive and red semi-active acceleration.	87
5.13	Carbody acceleration under sin wave input with frequency of 1Hz, yellow passive and red semi-active acceleration.	88
5.14	Carbody acceleration under sin wave input with frequency of 4Hz, yellow passive and red semi-active acceleration.	89
5.15	Carbody acceleration under sin wave input with frequency of 7Hz, yellow passive and red semi-active acceleration.	89
5.16	Carbody roll angle under random input, yellow passive and red semi-active roll angle.	90
5.17	Carbody roll angle under step input, yellow passive and red semi-active roll angle.	90
5.18	Carbody roll angle under sin wave input with frequency of 1Hz, yellow passive and red semi-active roll angle.	91
5.19	Carbody roll angle under sin wave input with frequency of 4Hz, yellow passive and red semi-active roll angle.	91
5.20	Carbody roll angle under sin wave input with frequency of 7Hz, yellow passive and red semi-active roll angle.	92
5.21	Carbody roll rate under random input, yellow passive and red semi-active body roll rate.	93

5.22	Carbody roll rate under step input, yellow passive and red semi-active body roll rate.	93
5.23	Carbody roll rate under sin wave input with frequency of 1Hz, yellow passive and red semi-active body roll rate.	94
5.24	Carbody roll rate under sin wave input with frequency of 4Hz, yellow passive and red semi-active body roll rate.	94
5.25	Carbody roll rate under sin wave input with frequency of 7Hz, yellow passive and red semi-active body roll rate.	95

LIST OF SYMBOLS

y_c	Carbody displacement
θ_c	Carbody roll angle
y_r	Relative displacement
y_b	Bogie Displacement
n_i	Number of cycles
a_i	Mean acceleration
T_i	Fatigue time
T	Total fatigue time
a	Peak acceleration
W_z	Sperling's Ride Index
f	Oscillation frequency
$F(f)$	Frequency-dependent factor
B	Acceleration weighting factor
B_w	Horizontal comfort index
B_s	Vertical comfort index
V_r	Vertical ride index
V_l	Lateral ride index
m_c	Mass of car body
m_b	Mass of car bogie
k_1	Secondary lateral spring stiffness
k_r	Stiffness of bogie disturbance
k_2	Secondary vertical spring stiffness
b_1	Secondary lateral damping coefficient
b_2	Secondary vertical damping coefficient
h_1	Height between body centre of gravity and secondary lateral suspension
w	Width of body centre gravity and secondary vertical suspension

CHAPTER 1

INTRODUCTION

Rail transportation has been the most demanded transport option offering safety, speed, and comfort. In parallel with implementation of new technologies, the cruising speed has also increased. The effects of vibrations caused by rail disturbances on vehicle carbody and passengers are more important in high cruising speeds. Hence, safe and comfortable transportation of passengers and goods under high speeds has become an important engineering problem to solve.

The vehicle suspension is used to eliminate unpleasant vibrations from various road conditions. There are three main types of vehicle suspension system have been effectively implemented. The systems are namely passive, semi-active and active systems. Though a passive suspension system featuring oil damper and spring provides design simplicity and cost-effectiveness, performance limitations are inevitable due to the lack of damping force controllability. On the other hand, an active suspension system can provides high control performance in wide frequency range. However, this type may require high power sources, many sensors and complex actuators such as servo valves. Consequently, one way to resolve these requirements of an active suspension system is to adopt a semi-active suspension system. The semi-active suspension system offers a desirable performance, enhanced in the active mode without requiring large power sources and expensive hardware.

Today's vehicles rely on a number of electronic control systems. Some of them are self-contained, stand-alone controllers fulfilling a particular function while others are co-ordinated by a higher-level supervisory logic. Examples of such vehicle control

systems include braking control, traction control, acceleration control, lateral stability control, suspension control and so forth. Such systems aim to enhance ride and handling, safety, driving comfort and driving pleasure. The thesis focuses on semi-active suspension control. The thrust of this work is to provide a comprehensive overview of modeling and design a vehicle semi-active systems based on smart damping devices. Isolation from the forces transmitted by external excitation is the fundamental task of any suspension system. The problem of mechanical vibration control is generally tackled by placing between the source of vibration and the structure to be protected, suspension systems composed of spring-type elements in parallel with dissipative elements. Suspensions are employed in mobile applications, such as vibrating machinery or civil structures. In the case of a vehicle, a classical car suspension aims to achieve isolation from the road by means of spring-type elements and viscous dampers (shock absorbers) and contemporarily to improve road holding and handling. The elastic element of a suspension is constituted by a spring (coil springs but also air springs and leaf springs), whereas the damping element is typically of the viscous type. In such a device the damping action is obtained by throttling a viscous fluid through orifices; depending on the physical properties of the fluid (mainly its viscosity), the geometry of the orifices and of the damper, a variety of force versus velocity characteristics can be obtained. This technology is very reliable and has been used since the beginning of the last century (Bastow, 1993).

1.1 Research Background

There has been a sustained interesting magneto-rheological (MR) device due to the controllable interface provided by the MR fluid inside the devices that enables the mechanical device to interact with an electronic system, which can be used to continuously adjust the mechanical properties of the device. Some examples of devices in which MR fluids have been employed include dampers, clutches, and brakes and transmissions.

The most popular of these devices are MR dampers, especially as automotive shock absorbers. The automotive shock absorber has been shown to be a very

important contributor to the ride comfort and road handling of a vehicle. It can conclude that the success of MR damper in semi-active vehicle suspension applications is determined by two aspects which is the accurate modeling of the MR dampers and the other is the selection of an appropriate control strategy.

In addition, theoretical and simulation researches have demonstrated that the performance of a semi-active control system is also highly dependent on the choice of control strategy. Therefore, some semi-active and passive control schemes have been discussed and compared the approaches, such as Stability Augmentation controller into semi-active control.

1.2 Objectives

- i. To design a controller for semi-active suspension system employing MR actuator for a secondary half car model of railway vehicle.
- ii. To tune the controller to investigate the desired performance of controller for body displacement and body acceleration of semi-active system using Genetic Algorithm methods.

1.3 Problem Statements

The suspension system must support the weight of the vehicle, provide directional control during handling maneuvers, and provide effective isolation of passengers and payload from disturbances.

A passive suspension has the ability to store energy via a spring and to dissipate it via a damper. The parameters are generally fixed, being chosen to achieve a certain level of stability and ride comfort. Once the spring has been selected based on the load-carrying capability of the suspension, the damper is the only variable remaining to specify. Low damping yields poor resonance control at the natural frequencies of the

body (sprung mass) and axle (unsprung mass), but provides the necessary high frequency isolation required for a comfortable ride. Conversely, large damping results in good resonance control at the expense of high frequency isolation. Due to these conflicting demands, suspension design has had to be something of a compromise, largely determined by the type of use for which the vehicle is designed.

The other solution is using active control. However this method is expensive for a standard train because require high power source, many sensors and complex actuator such as servo-valves. Consequently, one way to resolve this matter is to adopt the semi-active suspension system, where this system offers a desirable performance generally enhanced in the active mode without requiring large power sources and expensive hardware.

1.4 Research Question

Can Stability augmentation controller effectively control a semi-active suspension system leading to passengers' comfort?

1.5 Theoretical Frame Work

This study is to design and tune a stability controller to control a semi-active suspension system using half car model with MR damper.

1.6 Scopes of Research

- i. Modelling of semi-active suspension system using MR damper of a half car model within Matlab SIMULINK environment.
- ii. Genetic algorithm is implemented to tune the controller parameters.

- iii. Genetic algorithm to be implemented using Matlab and linked to SIMULINK.

1.7 Research Methodology and Flowchart

The methodologies involved in this study are shown in Figure 1.1. The project starts by collecting reading materials such as books, journals and technical papers specifically on railway vehicle model, passive, semi-active and active suspension system, MR damper, stability augmentation controller and evolutionary algorithm methods.

Research has been done continuously throughout this study to get a better understanding on the concept of semi-active suspension system and its constraints. Besides, consultation sessions with the project supervisor and few colleagues who are doing similar research were also held periodically to discuss any arising issues and problems encountered pertaining to this study.

Based on the research conducted, semi-active with MR damper application was crucially analyzed and its controller type were justified before used in simulation.

The study on a half-car railway vehicle suspension system has been divided into two main parts which are (1) mathematical modelling and (2) simulation of the controller system.

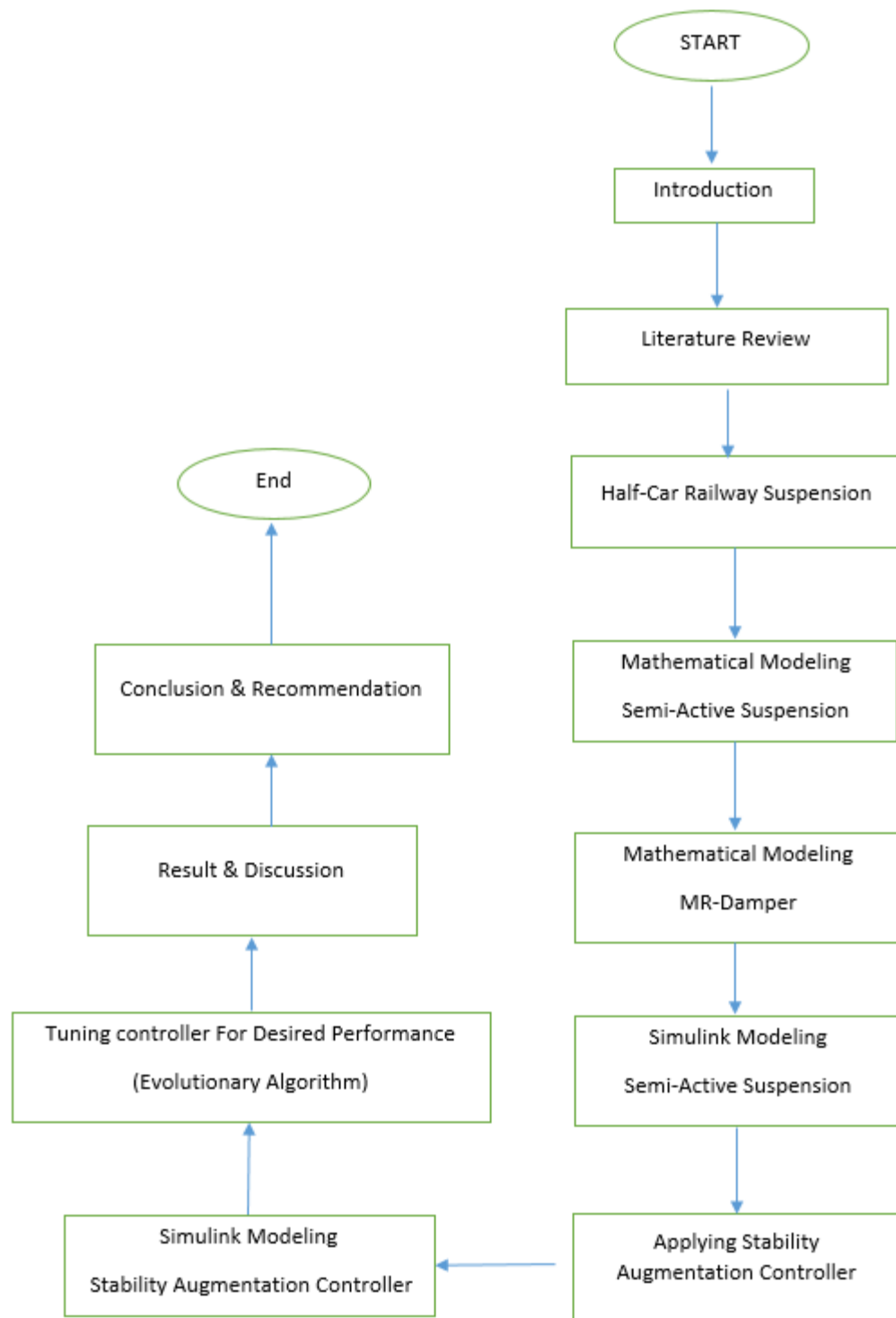


Figure 1.1 Flowchart of methodology

1.8 Thesis Outline

This thesis consists of six chapters. Chapter 1 is the introduction chapter. This chapter presents the research background, statement of the problem, objectives and scopes of the study, research contributions, methodology of research, and the overall outline of this thesis.

Chapter 2 presents the literature review on related subjects concerning this thesis. In this chapter, the classification of vehicle suspension system, stages, controllers, tuning methods for desired performance and review on published articles related to suspension control strategies are described.

Chapter 3 presents the modelling and validation of the half-car railway vehicle model. In this chapter, the mathematical equation of 3DOF half-car model is introduced. Other types of suspension systems will be described in detail. Then, the mathematical modeling of three different kinds of suspension system for half-car model and their SIMULINK model are presented in order to validate the simulation results. Two concepts of desired performance and their measurement methods will also be explained.

Chapter 4 describes the implementation of the proposed stability augmentation controller to achieve desired performance. In addition controller structure in SIMULINK and parameters are shown in SIMULINK. In this chapter, the fundamentals and algorithm of the proposed controller are explained.

Chapter 5 presents one real coded GA and explained in detail. After linking simulation model and GA code to tune the controller for the best performance is going to be done. In addition effects of GA parameters on the result will derived. At the end there is compresence part between two tuning method, sensitivity analysis and GA. Results for different inputs are presented and compared.

Finally, Chapter 6 is the concluding chapter. This chapter summarizes the works done in this entire study. The directions and recommendations for future research works are also outlined.

REFERENCES

- Alkhatib, R., Jazar, G. N., & Golnaraghi, M. F. (2004). Optimal design of passive linear suspension using genetic algorithm. *Journal of Sound and vibration*, 275(3), 665-691.
- Antonisse, J. A new interpretation of schema notation that overturns the binary encoding. In *Proceedings of the Third International Conference on Genetic Algorithms* (pp. 86-91).
- Bäck, T., & Hoffmeister, F. (1991). Extended selection mechanisms in genetic algorithms.
- Bäck, T., & Schwefel, H. P. (1993). An overview of evolutionary algorithms for parameter optimization. *Evolutionary computation*, 1(1), 1-23.
- Baker, J. E. (1985, July). Adaptive selection methods for genetic algorithms. In *Proceedings of an International Conference on Genetic Algorithms and their applications* (pp. 101-111).
- Baker, J. E. (1987, July). Reducing bias and inefficiency in the selection algorithm. In *Proceedings of the second international conference on genetic algorithms* (pp. 14-21).
- Batterbee, D. C., & Sims, N. D. (2007). Hardware-in-the-loop simulation of magnetorheological dampers for vehicle suspension systems. *Proceedings of the Institution of Mechanical Engineers, Part I: Journal of Systems and Control Engineering*, 221(2), 265-278.
- Batterbee, D. C., & Sims, N. D. (2007). Hardware-in-the-loop simulation of magnetorheological dampers for vehicle suspension systems. *Proceedings of the Institution of Mechanical Engineers, Part I: Journal of Systems and Control Engineering*, 221(2), 265-278.
- Baumal, A. E., McPhee, J. J., & Calamai, P. H. (1998). Application of genetic algorithms to the design optimization of an active vehicle suspension

- system. *Computer methods in applied mechanics and engineering*, 163(1), 87-94.
- BDack, T., Hoffmeister, F., & Schwefel, H. P. (1991, July). A survey of evolution strategies. In *Proceedings of the 4th international conference on genetic algorithms* (pp. 2-9).
- Berkovitz, L. D. (2013). *Optimal control theory* (Vol. 12). Springer Science & Business Media.
- Booker, L. B., Goldberg, D. E., & Holland, J. H. (1989). Classifier systems and genetic algorithms. *Artificial intelligence*, 40(1), 235-282.
- Bramlette, M. F. (1991, July). Initialization, Mutation and Selection Methods in Genetic Algorithms for Function Optimization. In *ICGA* (pp. 100-107).
- Bruni, S., Goodall, R., Mei, T. X., & Tsunashima, H. (2007). Control and monitoring for railway vehicle dynamics. *Vehicle System Dynamics*, 45(7-8), 743-779.
- Cebon, D., Besinger, F. H., & Cole, D. J. (1996). Control strategies for semi-active lorry suspensions. *Proceedings of the Institution of Mechanical Engineers, Part D: Journal of automobile engineering*, 210(2), 161-178.
- Chen, P. C., & Shih, M. C. (2007). Modeling and robust active control of a pneumatic vibration isolator. *Journal of Vibration and Control*, 13(11), 1553-1571.
- Chen, P. C., & Shih, M. C. (2011). Robust control of a novel active pneumatic vibration isolator through floor vibration observer. *Journal of Vibration and Control*, 17(9), 1325-1336.
- Chen, S., He, R., Liu, H., & Yao, M. (2012). Probe into necessity of active suspension based on LQG control. *Physics Procedia*, 25, 932-938.
- Choi, S. B., Lee, H. S., & Park, Y. P. (2002). H8 Control Performance of a Full-Vehicle Suspension Featuring Magnetorheological Dampers. *Vehicle System Dynamics*, 38(5), 341-360.
- Colombo, E. F., Di Gialleonardo, E., Facchinetti, A., & Bruni, S. (2014). Active carbody roll control in railway vehicles using hydraulic actuation. *Control Engineering Practice*, 31, 24-34.
- Cordón, O., & Herrera, O. F. (1993). A general study on genetic fuzzy systems.
- D'Amato, F. J., & Viassolo, D. E. (2000). Fuzzy control for active suspensions. *Mechatronics*, 10(8), 897-920.

- da Rocha, P. H., Ferreira, H. C., Porsch, M. C., & Sales, R. M. (2009). Fixed-point DSP implementation of nonlinear H_{∞} controller for large gap electromagnetic suspension system. *Control Engineering Practice*, 17(10), 1148-1156.
- da Rocha, P. H., Ferreira, H. C., Porsch, M. C., & Sales, R. M. (2009). Fixed-point DSP implementation of nonlinear H_{∞} controller for large gap electromagnetic suspension system. *Control Engineering Practice*, 17(10), 1148-1156.
- Davidor, Y. (1991). *Genetic Algorithms and Robotics: A heuristic strategy for optimization* (Vol. 1). World Scientific.
- Davis, L. (1989). Adapting operator probabilities in genetic algorithms. In *proc. 3rd International conference on genetic algorithms* (pp. 61-69).
- Davis, L. (1991). Handbook of genetic algorithms.
- Dorf, R. C., & Bishop, R. H. (1995). *Modern Control Systems*. Addison-Wesley Publishing Company.
- Dyke, S. J. (1996). *Acceleration feedback control strategies for active and semi-active control systems: modeling, algorithm development, and experimental verification*.
- Dyke, S. J., Spencer Jr, B. F., Sain, M. K., & Carlson, J. D. (1996, September). Experimental verification of semi-active structural control strategies using acceleration feedback. In *Proc. of the 3rd Intl. Conf. on Motion and Vibr. Control* (Vol. 3, pp. 291-296).
- Eberhart, R. C., & Shi, Y. (2001). Particle swarm optimization: developments, applications and resources. In *Evolutionary Computation, 2001. Proceedings of the 2001 Congress on* (Vol. 1, pp. 81-86). IEEE.
- Ehrgott, R. C., & Masri, S. F. (1994). Structural control applications of an electrorheological device. In *Proc., Int. Workshop on Struct. Control, USC Publ. No. CE-9311* (pp. 115-129).
- Elmi, N., Ohadi, A., & Samadi, B. (2013). Active front-steering control of a sport utility vehicle using a robust linear quadratic regulator method, with emphasis on the roll dynamics. *Proceedings of the Institution of Mechanical Engineers, Part D: Journal of automobile engineering*, 0954407013502319.
- Esat, I. I., Saud, M., & Engin, S. N. (2011). A novel method to obtain a real-time control force strategy using genetic algorithms for dynamic systems subjected to external arbitrary excitations. *Journal of Sound and Vibration*, 330(1), 27-48.

- Fagerlund, J. (2009). Towards Active Car Body Suspension in Railway Vehicles.
- Fagerlund, J., Sjöberg, J., & Abrahamsson, T. (2005). Passive Railway Car Secondary Suspension-Force, Power, Deflections, Roll and Comfort.
- Fischer, D., & Isermann, R. (2004). Mechatronic semi-active and active vehicle suspensions. *Control engineering practice*, 12(11), 1353-1367.
- Forrest, S. (1993). Genetic algorithms: principles of natural selection applied to computation. *Science*, 261(5123), 872-878.
- Fox, B. R., & McMahon, M. B. (1990). Genetic operators for sequencing problems. *Foundations of genetic algorithms*, 1, 284-300..
- Ghaedi, M., Ghaedi, A. M., Abdi, F., Roosta, M., Vafaei, A., & Asghari, A. (2013). Principal component analysis-adaptive neuro-fuzzy inference system modeling and genetic algorithm optimization of adsorption of methylene blue by activated carbon derived from Pistacia khinjuk. *Ecotoxicology and environmental safety*, 96, 110-117.
- Goldberg, D. E. (1990). Real-coded genetic algorithms, virtual alphabets, and blocking. *Urbana*, 51, 61801.
- Goldberg, D. E. (1994). Genetic and evolutionary algorithms come of age. *Communications of the ACM*, 37(3), 113-119.
- Guclu, R., & Metin, M. (2009). Fuzzy logic control of vibrations of a light rail transport vehicle in use in Istanbul traffic. *Journal of Vibration and Control*.
- Gysen, B. L., Paulides, J. J., Janssen, J. L., & Lomonova, E. (2010). Active electromagnetic suspension system for improved vehicle dynamics. *Vehicular Technology, IEEE Transactions on*, 59(3), 1156-1163.
- Habibi, H., Shirazi, K. H., & Shishesaz, M. (2008). Roll steer minimization of McPherson-strut suspension system using genetic algorithm method. *Mechanism and Machine Theory*, 43(1), 57-67.
- Hansen, N., & Ostermeier, A. (1996, May). Adapting arbitrary normal mutation distributions in evolution strategies: The covariance matrix adaptation. In *Evolutionary Computation, 1996., Proceedings of IEEE International Conference on* (pp. 312-317). IEEE.
- Hansen, N., & Ostermeier, A. (2001). Completely derandomized self-adaptation in evolution strategies. *Evolutionary computation*, 9(2), 159-195.
- Harun, M. H., Abdullah, W., Jamaluddin, H., Rahman, R. A., & Hudha, K. (2014, November). Hybrid skyhook-stability augmentation system for ride quality

- improvement of railway vehicle. In *Applied Mechanics and Materials*(Vol. 663, pp. 141-145).
- Herrera, F., Herrera-Viedma, E., Lozano, M., & Verdegay, J. L. (1994, September). Fuzzy tools to improve genetic algorithms. In *Second European Congress on Intelligent Techniques and Soft Computing* (Vol. 3, pp. 1532-1539).
- Herrera, F., Lozano, M., & Verdegay, J. L. (1995). Tuning fuzzy logic controllers by genetic algorithms. *International Journal of Approximate Reasoning*, 12(3), 299-315.
- Hertz, H. (1882). *Gesammelte Werke*, I, Leipzig (1895). Also *Ueber die Beruehrung festes elastisches Koerper, Jr. fuer Mathematik*, 92, 156.
- Hinterding, R., Michalewicz, Z., & Eiben, A. E. (1997, April). Adaptation in evolutionary computation: A survey. In *Evolutionary Computation, 1997., IEEE International Conference on* (pp. 65-69). IEEE.
- Hocking, L. M. (1991). *Optimal control: an introduction to the theory with applications*. Oxford University Press.
- Hou, K., Kalousek, J., & Dong, R. (2003). A dynamic model for an asymmetrical vehicle/track system. *Journal of Sound and Vibration*, 267(3), 591-604.
- Hrovat, D., & Sun, J. (1997). Models and control methodologies for IC engine idle speed control design. *Control Engineering Practice*, 5(8), 1093-1100.
- Hudha, K., Jamaluddin, H., & Samin, P. M. (2008). Disturbance rejection control of a light armoured vehicle using stability augmentation based active suspension system. *International Journal of Heavy Vehicle Systems*, 15(2-4), 152-169.
- Hudha, K., Jamaluddin, H., Samin, P. M., & Rahman, R. A. (2005). Effects of control techniques and damper constraint on the performance of a semi-active magnetorheological damper. *International journal of vehicle autonomous systems*, 3(2-4), 230-252.
- Ichikawa, Y., & Ishii, Y. (1993). Retaining diversity of genetic algorithms for multivariable optimization and neural network learning. In *Neural Networks, 1993., IEEE International Conference on* (pp. 1110-1114). IEEE.
- Jahromi, A. F., & Zabihollah, A. (2010, July). Linear quadratic regulator and fuzzy controller application in full-car model of suspension system with magnetorheological shock absorber. In *Mechatronics and Embedded Systems and Applications (MESA), 2010 IEEE/ASME International Conference on* (pp. 522-528). IEEE.

- Jamaluddin, H., Bakar, A., Anuar, S., & Hudha, K. (2008). Semi-active suspension for ride improvement using stability augmentation system control algorithm. *Jurnal Mekanikal*, (26), 86-95.
- Karthikraja, A., Petchinathan, G., & Ramesh, S. (2009, June). Stochastic algorithm for PID tuning of bus suspension system. In *Control, Automation, Communication and Energy Conservation, 2009. INCACEC 2009. 2009 International Conference on* (pp. 1-6). IEEE.
- Kennedy, J., Kennedy, J. F., Eberhart, R. C., & Shi, Y. (2001). *Swarm intelligence*. Morgan Kaufmann.
- Kjellqvist, P. (2002). Modelling and design of electromechanical actuators for active suspension in rail vehicles.
- Kjellqvist, P. (2002). Modelling and design of electromechanical actuators for active suspension in rail vehicles.
- Koza, J. R. (1992). *Genetic programming: on the programming of computers by means of natural selection* (Vol. 1). MIT press.
- Lanchester, F. W. (1936). *The Theory of Dimensions and Its Applications for Engineers*. Taylor & Francis, Limited.
- Lee, H. S., & Choi, S. B. (2000). Control and response characteristics of a magneto-rheological fluid damper for passenger vehicles. *Journal of Intelligent Material Systems and Structures*, 11(1), 80-87.
- Liepins, G. E., & Vose, M. D. (1992). Characterizing crossover in genetic algorithms. *Annals of Mathematics and Artificial Intelligence*, 5(1), 27-34.
- Lin, R. C., Cebon, D., & Cole, D. J. (1996). Optimal roll control of a single-unit lorry. *Proceedings of the Institution of Mechanical Engineers, Part D: Journal of Automobile Engineering*, 210(1), 45-55.
- Lou, Z., Ervin, R. D., & Filisko, F. E. (1994). A preliminary parametric study of electrorheological dampers. *Journal of fluids engineering*, 116(3), 570-576.
- Lucasius, C. B., & Kateman, G. (1989, December). Application of genetic algorithms in chemometrics. In *Proceedings of the third international conference on Genetic algorithms* (pp. 170-176). Morgan Kaufmann Publishers Inc..
- Marzbanrad, J., Poozesh, P., & Damroodi, M. (2013). Improving vehicle ride comfort using an active and semi-active controller in a half-car model. *Journal of Vibration and Control*, 19(9), 1357-1377.
- Mashadi, B., Mostaani, S., & Majidi, M. (2011). Vehicle stability enhancement by using an active differential. *Proceedings of the Institution of Mechanical*

Engineers, Part I: Journal of Systems and Control Engineering, 0959651811405113.

- Metered, H., Bonello, P., & Oyadiji, S. O. (2010). The experimental identification of magnetorheological dampers and evaluation of their controllers. *Mechanical Systems and Signal Processing*, 24(4), 976-994.
- Metin, M., & Guclu, R. (2014). Rail vehicle vibrations control using parameters adaptive PID controller. *Mathematical Problems in Engineering*, 2014.
- Metin, M., Guclu, R., Yazici, H., & Yalcin, N. S. (2008, June). A comparison of control algorithms for a half rail vehicle model under track irregularity effect. In *Mechanics Conference and 100th Anniversary of Engineering of Science and Mechanics Department Virginia Polytechnic and State University*.
- Michaelwicz, Z. (1994). Genetic Algorithms+ Data Structures= Evolution Programs Springer. New York, 1, 992.
- Milliken, W. F., & Milliken, D. L. (1995). *Race car vehicle dynamics* (Vol. 400). Warrendale: Society of Automotive Engineers.
- Mirzaei, M., & Hassannejad, R. (2007). Application of genetic algorithms to optimum design of elasto-damping elements of a half-car model under random road excitations. *Proceedings of the Institution of Mechanical Engineers, Part K: Journal of Multi-body Dynamics*, 221(4), 515-526.
- Modares, H., Alfi, A., & Fateh, M. M. (2010). Parameter identification of chaotic dynamic systems through an improved particle swarm optimization. *Expert Systems with Applications*, 37(5), 3714-3720.
- Nguyen, Q. H., & Choi, S. B. (2009). Optimal design of MR shock absorber and application to vehicle suspension. *Smart materials and Structures*, 18(3), 035012.
- Pacchioni, A., Goodall, R. M., & Bruni, S. (2010). Active suspension for a two-axle railway vehicle. *Vehicle System Dynamics*, 48(S1), 105-120.
- Passino, K. M., Yurkovich, S., & Reinfrank, M. (1998). *Fuzzy control* (Vol. 42, pp. 15-21). Menlo Park, CA: Addison-wesley.
- Pearson, J. T., Goodall, R. M., & Pratt, I. (1998). Control system studies of an active anti-roll bar tilt system for railway vehicles. *Proceedings of the Institution of Mechanical Engineers, Part F: Journal of Rail and Rapid Transit*, 212(1), 43-60.

- Pearson, J. T., Goodall, R. M., & Pratt, I. (1998). Control system studies of an active anti-roll bar tilt system for railway vehicles. *Proceedings of the Institution of Mechanical Engineers, Part F: Journal of Rail and Rapid Transit*, 212(1), 43-60.
- Pearson, J. T., Goodall, R. M., & Pratt, I. (1998). Control system studies of an active anti-roll bar tilt system for railway vehicles. *Proceedings of the Institution of Mechanical Engineers, Part F: Journal of Rail and Rapid Transit*, 212(1), 43-60.
- Persson, R. (2007). Tilting trains: description and analysis of the present situation.
- Persson, R. (2007). Tilting trains: description and analysis of the present situation.
- Petchinathan, A. K. G., & Ramesh, S. PID Based Bus Suspension System Control Using Evolutionary Algorithms.
- Poli, R., Kennedy, J., & Blackwell, T. (2007). Particle swarm optimization. *Swarm intelligence*, 1(1), 33-57.
- Priyandoko, G., Mailah, M., & Jamaluddin, H. (2009). Vehicle active suspension system using skyhook adaptive neuro active force control. *Mechanical Systems and Signal Processing*, 23(3), 855-868.
- Pugi, L., Rindi, A., Bartolini, F., & Auciello, J. (2009, August). Simulation of a semi-active suspension system for a high speed train. In *Proceedings of the International Symposium on Dynamics of Vehicles on Roads and Tracks (IAVSD'09)*, Stockholm, Sweden.
- Radcliffe, N. J. (1992). Non-Linear Genetic Representations. In *PPSN* (pp. 261-270).
- ROTH, P. A., & LIZELL, M. (1996). A lateral semi-active damping system for trains. *Vehicle System Dynamics*, 25(S1), 585-598.
- Sanner, R. M., & Slotine, J. J. E. (1992). Gaussian networks for direct adaptive control. *Neural Networks, IEEE Transactions on*, 3(6), 837-863.
- Sassi, S., Cherif, K., Mezghani, L., Thomas, M., & Kotrane, A. (2005). An innovative magnetorheological damper for automotive suspension: from design to experimental characterization. *Smart Materials and Structures*, 14(4), 811.
- Schlierkamp-Voosen, D., & Mühlenbein, H. (1993). Predictive models for the breeder genetic algorithm. *Evolutionary Computation*, 1(1), 25-49.
- Schwefel, H. P. (1995). Evolution and Optimum Seeking. Sixth-Generation Computer Technology Series.

- Sezer, S., & Atalay, A. E. (2011). Dynamic modeling and fuzzy logic control of vibrations of a railway vehicle for different track irregularities. *Simulation Modelling Practice and Theory*, 19(9), 1873-1894.
- Sezer, S., & Atalay, A. E. (2012). Application of fuzzy logic based control algorithms on a railway vehicle considering random track irregularities. *Journal of Vibration and Control*, 18(8), 1177-1198.
- Shimamune, R. Y., & Tanifuji, K. (1995, July). Application of oil-hydraulic actuator for active suspension of railway vehicle: experimental study. In *SICE'95. Proceedings of the 34th SICE Annual Conference. International Session Papers* (pp. 1335-1340). IEEE.
- Shin, Y. J., You, W. H., Hur, H. M., Park, J. H., & Lee, G. S. (2014). Improvement of Ride Quality of Railway Vehicle by Semi-active Secondary Suspension System on Roller Rig Using Magnetorheological Damper. *Advances in Mechanical Engineering*, 6, 298382.
- Sohn, H. C., Hong, K. S., & Hedrick, J. K. (2000). Semi-active control of the Macpherson suspension system: hardware-in-the-loop simulations. In *Control Applications, 2000. Proceedings of the 2000 IEEE International Conference on* (pp. 982-987). IEEE.
- Spears, W. M. (2013). *Evolutionary algorithms: the role of mutation and recombination*. Springer Science & Business Media.
- Sugahara, Y., Kazato, A., Koganei, R., Sampei, M., & Nakaura, S. (2009). Suppression of vertical bending and rigid-body-mode vibration in railway vehicle car body by primary and secondary suspension control: results of simulations and running tests using Shinkansen vehicle. *Proceedings of the Institution of Mechanical Engineers, Part F: Journal of Rail and Rapid Transit*, 223(6), 517-531.
- SUGAHARA, Y., KAZATO, A., TAKIGAMI, T., & KOGANEI, R. (2008). Suppression of vertical vibration in railway vehicles by controlling the damping force of primary and secondary suspensions. *Quarterly Report of RTRI*, 49(1), 7-15.
- Sugahara, Y., Takigami, T., Kazato, A., Koganei, R., & Sampei, M. (2008). Suppression of vertical vibration in railway vehicles by damping force control of primary suspension using an LQG controller. *Journal of System Design and Dynamics*, 2(1), 251-262.

- Sugahara, Y., Watanabe, N., Takigami, T., & Koganei, R. (2011). Vertical Vibration Suppression System for Railway Vehicles Based on Primary Suspension Damping Control-System Development and Vehicle Running Test Results. *Quarterly Report of RTRI*, 52(1), 13-19.
- Sun, L., Cai, X., & Yang, J. (2007). Genetic algorithm-based optimum vehicle suspension design using minimum dynamic pavement load as a design criterion. *Journal of Sound and Vibration*, 301(1), 18-27.
- Sun, Y. Q., & Simson, S. (2007). Nonlinear three-dimensional wagon-track model for the investigation of rail corrugation initiation on curved track. *Vehicle System Dynamics*, 45(2), 113-132.
- Syswerda, G. (1991). Schedule optimization using genetic algorithms. *Handbook of genetic algorithms*.
- Tang, C., & Zhang, T. (2005, October). The research on control algorithms of vehicle active suspension system. In *Vehicular Electronics and Safety, 2005. IEEE International Conference on* (pp. 320-325). IEEE.
- Tung, S. L., Juang, Y. T., Lee, W. H., Shieh, W. Y., & Wu, W. Y. (2011). Optimization of the exponential stabilization problem in active suspension system using PSO. *Expert Systems with Applications*, 38(11), 14044-14051.
- Ulsoy, A. G., & Hrovat, D. (1990). Stability robustness of LQG active suspensions. In *1990 American Control Conference* (pp. 1347-1356).
- Vignaux, G. A., & Michalewicz, Z. (1991). A genetic algorithm for the linear transportation problem. *Systems, Man and Cybernetics, IEEE Transactions on*, 21(2), 445-452.
- Voigt, H. M. (1992). *Fuzzy evolutionary algorithms*. International Computer Science Institute.
- Voigt, H. M., Born, J., & Santibáñez-Koref, I. (1993). *A multivalued evolutionary algorithm*. International Computer Science Institute.
- Watton, J., Holford, K. M., & Surawattanawan, P. (2004). The application of a programmable servo controller to state control of an electrohydraulic active suspension. *Proceedings of the Institution of Mechanical Engineers, Part D: Journal of Automobile Engineering*, 218(12), 1367-1377.
- Whitley, L. D., Starkweather, T., & D'Ann Fuquay. (1989, June). Scheduling problems and traveling salesmen: The genetic edge recombination operator. In *ICGA* (Vol. 89, pp. 133-40).

- Williams, R. A. (1997). Automotive active suspensions Part 2: Practical considerations. *Proceedings of the Institution of Mechanical Engineers, Part D: Journal of Automobile Engineering*, 211(6), 427-444.
- Yagiz, N., Hacioglu, Y., & Taskin, Y. (2008). Fuzzy sliding-mode control of active suspensions. *Industrial Electronics, IEEE Transactions on*, 55(11), 3883-3890.
- Yang, G., Chen, R., Yin, C., Gao, J., Zhang, L., Wu, Y., ... & Li, J. (2010). Available online at www.sciencedirect.com. *Neurosci. Lett*, 485, 138-142.
- Zapateiro, M., Pozo, F., Karimi, H. R., & Luo, N. (2012). Semi-active control methodologies for suspension control with magnetorheological dampers. *Mechatronics, IEEE/ASME Transactions on*, 17(2), 370-380.
- Zhou, J., Goodall, R., Ren, L., & Zhang, H. (2009). Influences of car body vertical flexibility on ride quality of passenger railway vehicles. *Proceedings of the Institution of Mechanical Engineers, Part F: Journal of Rail and Rapid Transit*, 223(5), 461-471.
- Zhou, R., Zolotas, A., & Goodall, R. M. (2009). Integrated tilt and active lateral secondary suspension control.
- Ziegler, J. G., & Nichols, N. B. (1942). Optimum settings for automatic controllers. *trans. ASME*, 64(11).
- Zolotas, A. C., & Goodall, R. M. (2007). Modelling and control of railway vehicle suspensions. In *Mathematical Methods for Robust and Nonlinear Control* (pp. 373-411). Springer London.